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HARMONICALLY EXCITED VIBRATIONS

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Dynamics and Oscillations

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**Partnership for Promotion and Popularization of Electrical Mobility through
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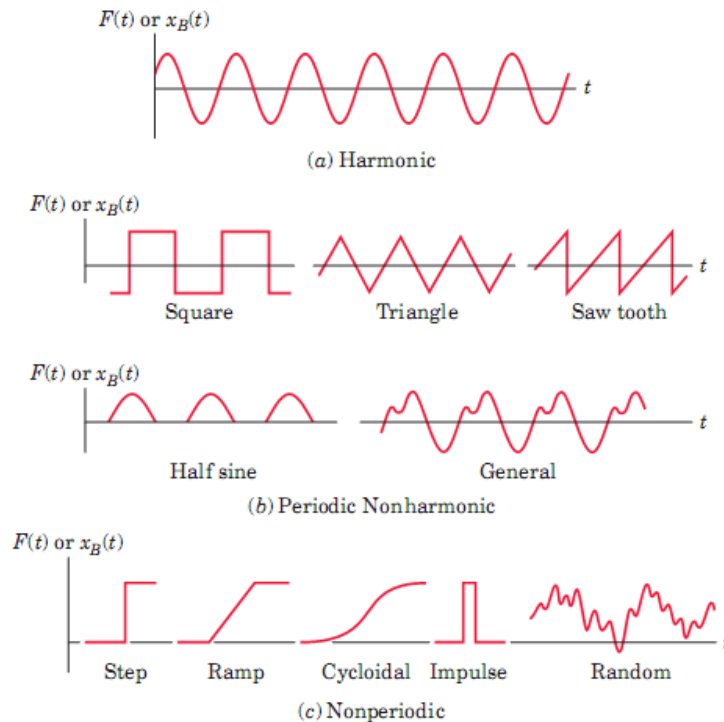
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Types of disturbance forces

If there is present some disturbance force that impacts on free harmonic vibration motion, then that motion is called **FORCED VIBRATION**.

DISTURBANCE FORCE TYPES:

- ***Harmonic***
- ***Periodical nonharmonic***
- ***Nonperiodic***



Equation of motion of harmonically excited vibrations

$$m\ddot{x} = -kx + F_0 \sin(\Omega t + \beta)$$

$$m\ddot{x} + kx = F_0 \sin(\Omega t + \beta)$$

$$\ddot{x} + \omega^2 x = h \sin(\Omega t + \beta)$$

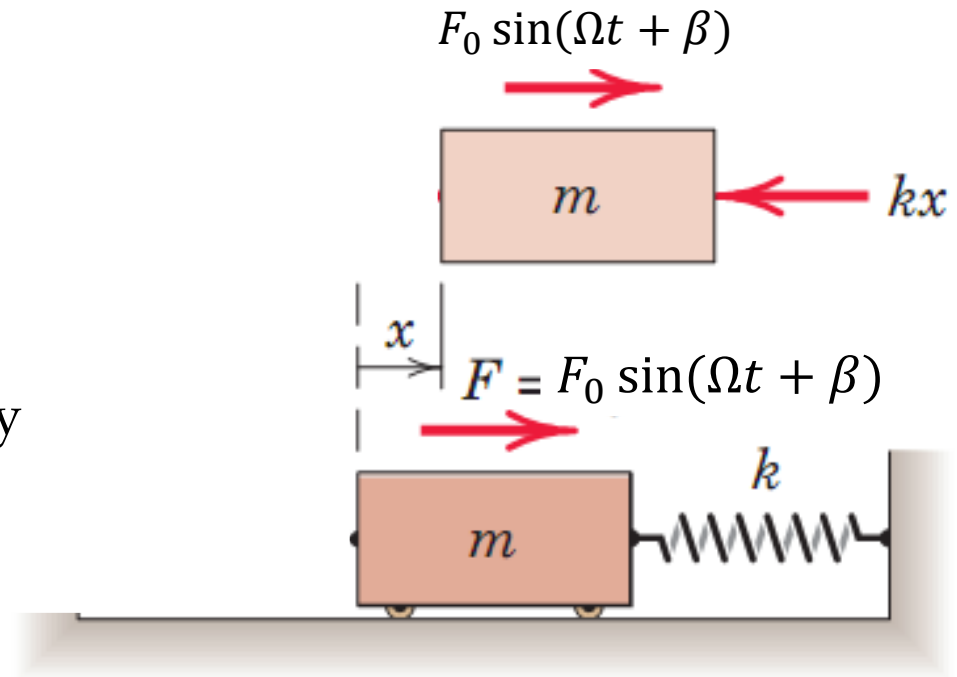
Differential equation of harmonically excited vibrations

Ω – exciting force natural frequency

F_0 – exciting force amplitude

β – exciting force phase shift

$$h = \frac{F_0}{m}$$



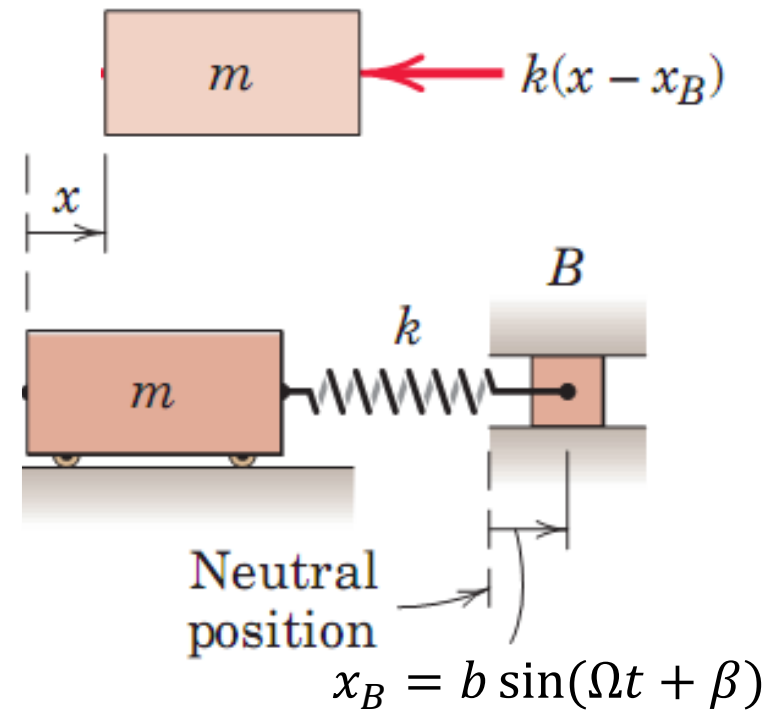
Harmonic motion of base

$$m\ddot{x} = -k(x - x_B)$$

$$\ddot{x} + \omega^2 x = \frac{kb}{m} \sin(\Omega t + \beta)$$

*Differential equation of
undamped forced
vibration – base motion
case*

b – base movement amplitude



Solution of the equation of motion

$$x = x_h + x_p$$

$$x_h = C_1 \cos \omega t + C_2 \sin \omega t$$

$$\omega \neq \Omega$$

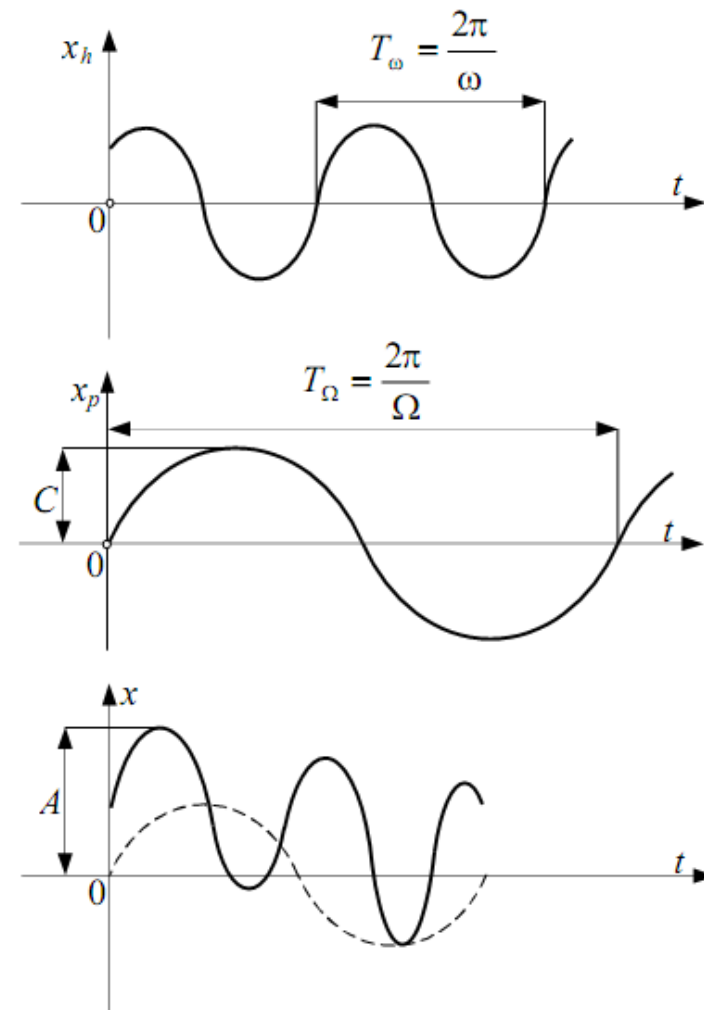
$$x_p = C \sin(\Omega t + \beta) \quad \text{where} \quad C = \frac{h}{\omega^2 - \Omega^2}$$

$$x_p = \frac{h}{\omega^2 - \Omega^2} \sin(\Omega t + \beta)$$

$$x = C_1 \cos \omega t + C_2 \sin \omega t + \frac{h}{\omega^2 - \Omega^2} \sin(\Omega t + \beta)$$

or

$$x = A \sin(\omega t + \alpha) + \frac{h}{\omega^2 - \Omega^2} \sin(\Omega t + \beta)$$



Resonance ($\Omega = \omega$)

$$x_p = \frac{\frac{h}{\omega^2}}{1 - \left(\frac{\Omega}{\omega}\right)^2} \sin(\Omega t + \beta)$$

Statical spring elongation

$$\frac{h}{\omega^2} = \frac{m \cdot h}{c} = f_{st}$$

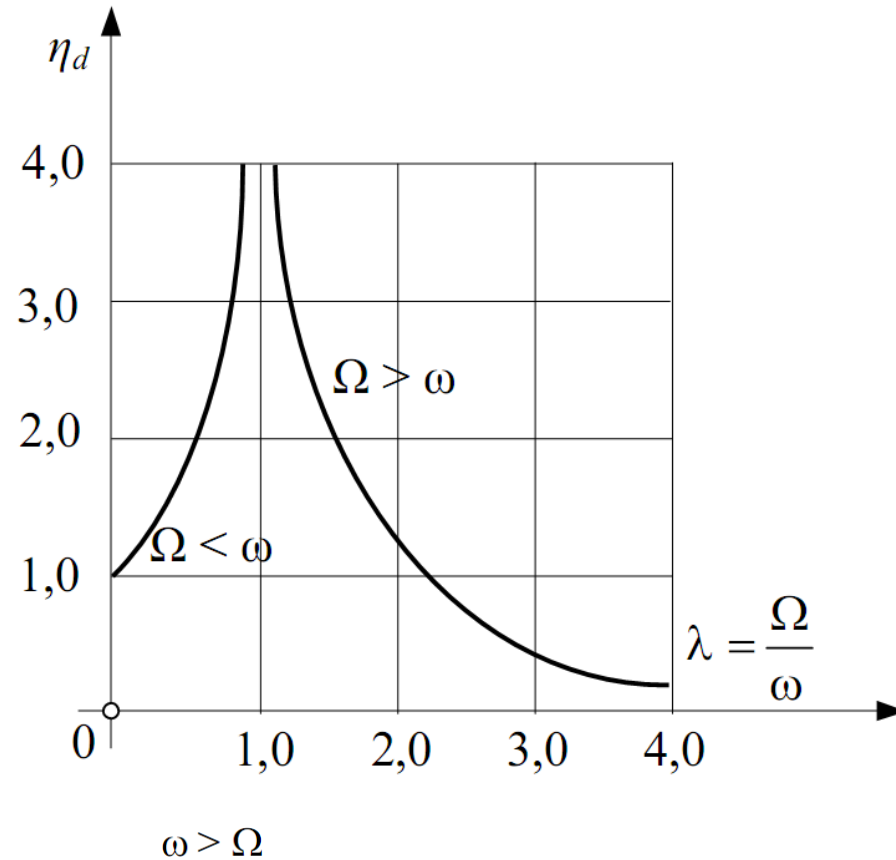
Disturbance coefficient

$$\lambda = \frac{\Omega}{\omega}$$

Amplification factor

$$\eta_d = \frac{C_d}{C_{st}} = \frac{\frac{h}{\omega^2 - \Omega^2}}{f_{st}} = \frac{\omega^2}{\omega^2 - \Omega^2} = \frac{\omega^2}{\omega^2 \left[1 - \left(\frac{\Omega}{\omega}\right)^2 \right]} = \frac{1}{1 - \lambda^2}$$

$$\eta_d = \frac{1}{\lambda^2 - 1} \quad \omega < \Omega$$



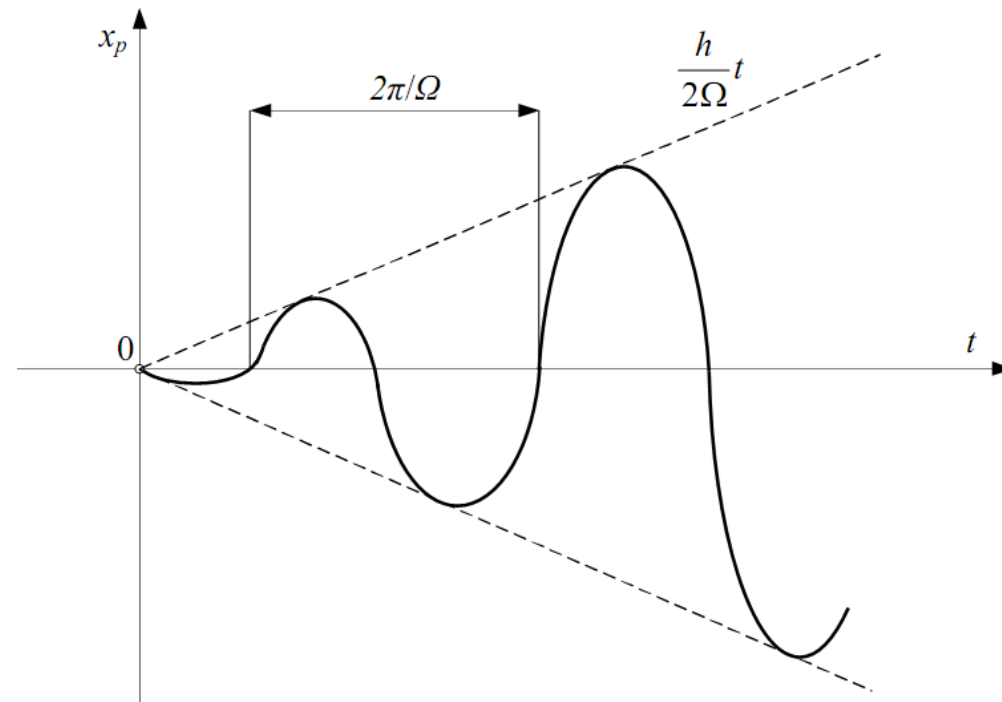
$$x_p = A t \cos(\Omega t) + B t \sin(\omega t)$$

$$A = -\frac{h}{2\Omega} \quad B = 0$$

$$x_p = -\frac{h}{2\Omega} t \cos(\Omega t).$$

Equation of motion for resonance

$(\Omega = \omega)$



Beating

Beating as phenomena would appear if frequency of the system and exciting force are nearly equal.

During the beating phase of those two movements are changing from 0 to 2π .

$$x = C_1 \cos(\omega t) + C_2 \sin(\omega t) + \frac{h}{\omega^2 - \Omega^2} \sin(\Omega t)$$

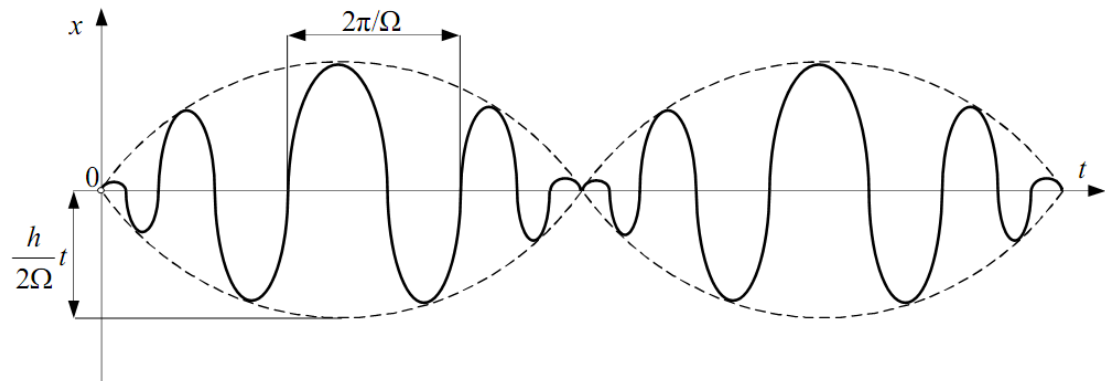
$$0 = C_1, \quad 0 = C_2 \omega + \frac{h}{\omega^2 - \Omega^2}$$

$$\omega - \Omega = 2\Delta$$

$$\frac{\Omega}{\omega} = \frac{\omega - 2\Delta}{\omega} \approx 1$$

$$x = \frac{h}{4\Omega\Delta} \cos(\Omega t) \sin(\Delta t)$$

$$\frac{\Omega}{\omega} \approx 1$$



Equation of motion for beating

Damped forced motion

$$m\ddot{x} + c\dot{x} + kx = F_0 \sin(\Omega t + \beta)$$

$$\ddot{x} + 2\delta\dot{x} + \omega^2 x = h \cdot \sin(\Omega t + \beta)$$

$$x_h = e^{-\delta t} (C_1 \cos pt + C_2 \sin pt)$$

$$x_p = C \sin(\Omega t + \beta - \varphi_0)$$

$$C = \frac{h}{\sqrt{(\omega^2 - \Omega^2)^2 + 4\delta^2 \Omega^2}}$$

$$\tan \varphi_0 = \frac{2\delta\Omega}{\omega^2 - \Omega^2}$$

$$x = e^{-\delta t} (C_1 \cos pt + C_2 \sin pt) + C \sin(\Omega t + \beta - \varphi_0)$$

Equation of motion for damped forced motion

$\lambda = \frac{\Omega}{\omega}$ · Dimensionless disturbance coefficient

$\psi = \frac{\delta}{\omega}$ · Dimensionless damping coefficient

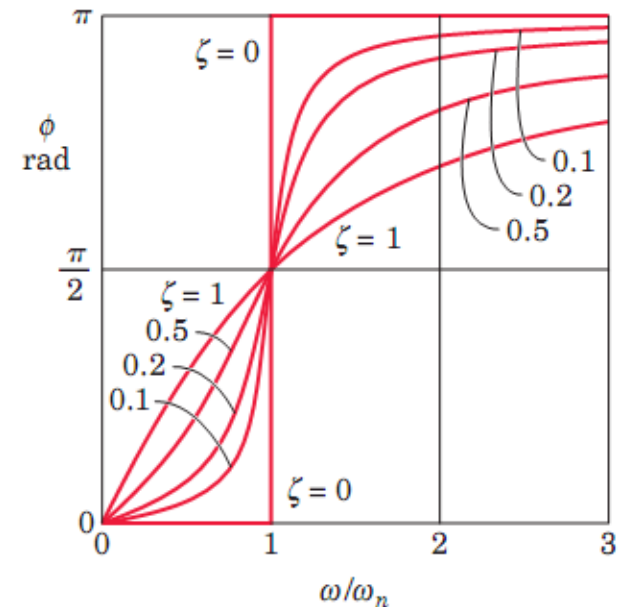
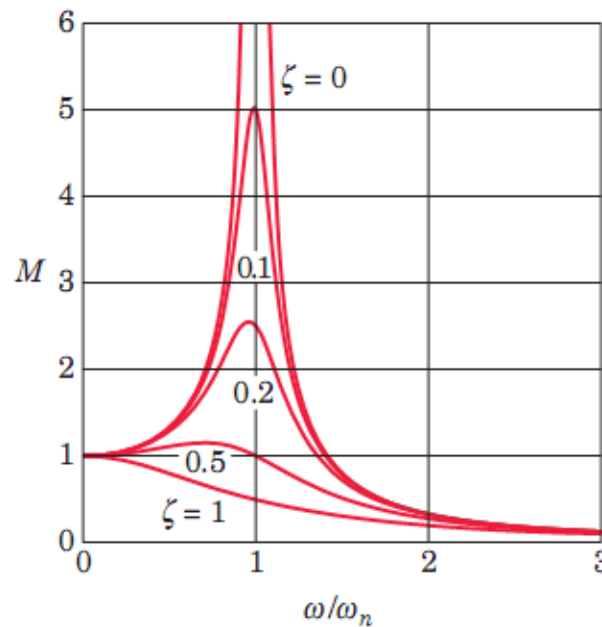
$$C = \frac{h}{\omega^2 \sqrt{(1 - \lambda^2)^2 + 4\psi^2 \lambda^2}}$$

$$\frac{h}{\omega^2} = f_{st} = C_{st}$$

$$\varphi_0 = \arctg \frac{2\lambda\psi}{1 - \lambda^2}$$

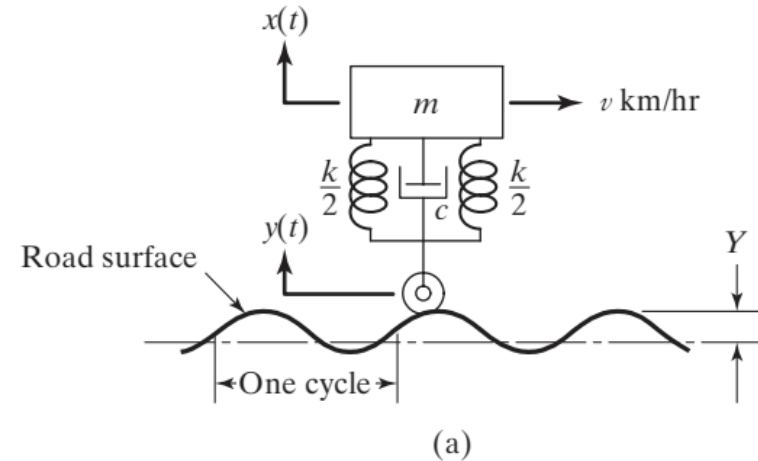
$$\eta_d = \frac{C}{C_{st}} = \frac{1}{\sqrt{(1 - \lambda^2)^2 + 4\psi^2 \lambda^2}}$$

Amplification factor
of damped forced vibrations



EXAMPLE:

Figure shows a simple model of a small electrical motor vehicle that can vibrate in the vertical direction while traveling over a rough road. The vehicle has a mass of 1200 kg. The suspension system has a spring constant of 400 kN/m and a damping ratio of $\zeta = 0.5$. If the vehicle speed is 20 km/hr, determine the displacement amplitude of the vehicle. The road surface varies sinusoidally with an amplitude of $Y = 0.05$ m and a wavelength of 6 m.



SOLUTION:

$$\omega = 2\pi f = 2\pi \left(\frac{v \times 1000}{3600} \right) \frac{1}{6} = 0.290889v \text{ rad/s}$$

$$\omega_n = \sqrt{\frac{k}{m}} = \left(\frac{400 \times 10^3}{1200} \right)^{1/2} = 18.2574 \text{ rad/s}$$

$$\frac{X}{Y} = \left\{ \frac{1 + (2\zeta r)^2}{(1 - r^2)^2 + (2\zeta r)^2} \right\}^{1/2} = \left\{ \frac{1 + (2 \times 0.5 \times 0.318653)^2}{(1 - 0.318653)^2 + (2 \times 0.5 \times 0.318653)^2} \right\}^{1/2}$$

$$= 1.100964$$

$$X = 1.100964Y = 1.100964(0.05) = 0.055048 \text{ m}$$

$$r = \frac{\omega}{\omega_n} = \frac{5.81778}{18.2574} = 0.318653$$